

Heat and Mass Transfer (DI05019071)

Complete Lecture Deck

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Overall Heat Transfer Coefficient

- Course: Heat and Mass Transfer (DI05019071)
- Unit 2: Convection
- Lecture 18: Overall Heat Transfer Coefficient

Lecture Agenda

- 1. Real-world Heat Transfer Scenarios
- 2. The Thermal Resistance Concept Revisited
- 3. Defining Convection Resistance
- 4. Heat Transfer through a Plane Wall (Convection on both sides)
- 5. The Overall Heat Transfer Coefficient (U)
- 6. U for Cylindrical Geometries (Pipes)
- 7. Fouling and Thermal Contact Resistance

Real-world Scenarios

- In most engineering problems, heat transfer involves multiple modes in series.
- Example 1: Heat lost from a room through a wall to the cold outside air. (Inner convection - h_i Wall conduction - k Outer convection).
- Example 2: Heat exchanger tube. Hot fluid inside, cold fluid outside. (Inner convection - h_i Tube wall conduction - k Outer convection).

Thermal Resistance Revisited

- Recall from Unit 1: Heat transfer can be modeled analogously to electrical current.
- Current (I) - $\rightarrow$ Heat transfer rate (q)
- Voltage Difference (ΔV) - $\rightarrow$ Temperature Difference (ΔT)
- Electrical Resistance (R_e) - $\rightarrow$ Thermal Resistance (R_t)
- $q = \Delta T / R_{\text{total}}$

Convection Resistance

- Newton's Law of Cooling: $q = h * A * (T_s - T_{\infty})$
- Rearranging into the resistance format ($q = \Delta T / R$):
- $q = (T_s - T_{\infty}) / (1 / (h * A))$
- Therefore, Convection Thermal Resistance is:
- $R_{\text{conv}} = 1 / (h * A)$

Heat Transfer through a Plane Wall

- Consider a wall of thickness 'L', thermal conductivity 'k', area 'A'.
- Hot fluid (T_h , h_1) on one side, cold fluid (T_c , h_2) on the other.
- Thermal circuit consists of three resistances in series:
 - 1. $R_{\text{conv1}} = 1 / (h_1 * A)$
 - 2. $R_{\text{cond}} = L / (k * A)$
 - 3. $R_{\text{conv2}} = 1 / (h_2 * A)$

The Overall Heat Transfer Coefficient (U)

- To simplify calculations, we define the Overall Heat Transfer Coefficient (U).
- Equation: $q = U * A * \Delta T_{\text{overall}}$
- Where $\Delta T_{\text{overall}} = T_{\text{hot_fluid}} - T_{\text{cold_fluid}}$.
- Comparing with $q = \Delta T / R_{\text{total}}$, we get:
- $U * A = 1 / R_{\text{total}} \Rightarrow U = 1 / (R_{\text{total}} * A)$

U for a Plane Wall

- For our plane wall with convection on both sides:
- $R_{\text{total}} = (1 / h_1 A) + (L / k A) + (1 / h_2 A)$
- Since $U = 1 / (R_{\text{total}} * A)$, we can factor out 'A':
- $U = 1 / [(1/h_1) + (L/k) + (1/h_2)]$
- Units for U are $\text{W}/\text{m}^2 \cdot \text{K}$ (same as h).

U for Cylindrical Geometries

- For pipes, the inner area ($A_i = 2\pi r_i L$) is different from the outer area ($A_o = 2\pi r_o L$).
- Because A is not constant, U depends on which area it is based on:
- $U_i * A_i = U_o * A_o = 1 / R_{total}$
- $U_i = 1 / [(1/h_i) + (A_i \ln(r_o/r_i)/(2\pi kL)) + (A_i/(A_o * h_o))]$

Fouling Factor (R_f)

- Over time, surfaces accumulate deposits (scale, rust, algae).
- These deposits act as additional thermal insulation, reducing heat transfer.
- This added resistance is called the Fouling Factor (R_f).
- $R_{\text{total}} = R_{\text{conv1}} + R_{f1} + R_{\text{cond}} + R_{f2} + R_{\text{conv2}}$

Impact of Fouling on U

- Fouling always increases the total thermal resistance.
- Therefore, fouling always decreases the Overall Heat Transfer Coefficient (U).
- Equipment must be designed with extra capacity (a clean U value vs a dirty U value) to allow for fouling over its operational life.

Summary

- Convection thermal resistance is $R_{\text{conv}} = 1 / (hA)$.
- The Overall Heat Transfer Coefficient (U) combines all series resistances: $U \cdot A = 1/R_{\text{total}}$.
- For pipes, U must be specified relative to the inner or outer area.
- Fouling adds thermal resistance and degrades performance over time.

Next Lecture Preview

- Topic: Dimensionless numbers and their physical significance
- Key questions:
 - - How do we simplify complex convection problems?
 - - What are the Reynolds, Prandtl, and Nusselt numbers?