

Heat and Mass Transfer (DI05019071)

Complete Lecture Deck

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Overall Heat Transfer Coefficient

- Course: Heat and Mass Transfer (DI05019071)
- Unit 1: Conduction
- Lecture 10: Overall heat transfer coefficient

Lecture Agenda

- 1. Introduction to Combined Modes
- 2. Defining the Overall Heat Transfer Coefficient (U)
- 3. U for a Plane Wall
- 4. U for a Cylindrical Surface
- 5. Inner vs. Outer Area Basis for U
- 6. Fouling Factors

Combined Modes of Heat Transfer

- Most engineering problems involve multiple modes of heat transfer.
- Example: Hot fluid inside a pipe, cold air outside.
- Heat transfers by: Convection (inside) - $\rightarrow$ Conduction (through pipe wall) - $\rightarrow$ Convection (outside).
- We need a simple way to express this total process.

Defining the Overall Coefficient (U)

- We define the overall heat transfer coefficient, U , such that:
- $q = U * A * \Delta T_{\text{total}}$
- Where ΔT_{total} is the temperature difference between the two bulk fluids.
- Comparing this to $q = \Delta T_{\text{total}} / R_{\text{total}}$, we see that:
- $U * A = 1 / R_{\text{total}}$

U for a Plane Wall

- Consider a wall of thickness L , conductivity k , with fluids h_{in} and h_{out} .
- $R_{total} = (1/h_{in} \cdot A) + (L/k \cdot A) + (1/h_{out} \cdot A)$
- Since area A is constant: $R_{total} = (1/A) \cdot [1/h_{in} + L/k + 1/h_{out}]$
- Therefore: $1/U = 1/h_{in} + L/k + 1/h_{out}$

U for a Cylindrical Surface

- For a pipe, the inner area (A_{in}) is not equal to the outer area (A_{out}).
- $R_{total} = 1/(h_{in}A_{in}) + \ln(r_{out}/r_{in})/(2\pi Lk) + 1/(h_{out}A_{out})$
- Because area varies, U depends on which area we use as a reference.
- $q = U_{in} * A_{in} * \Delta T = U_{out} * A_{out} * \Delta T$

Inner vs. Outer Area Basis

- Based on inner area ($A_{in} = 2\pi r_{in}L$):
 - $1/U_{in} = 1/h_{in} + [r_{in} * \ln(r_{out}/r_{in})]/k + (r_{in}/r_{out})*(1/h_{out})$
- Based on outer area ($A_{out} = 2\pi r_{out}L$):
 - $1/U_{out} = (r_{out}/r_{in})*(1/h_{in}) + [r_{out} * \ln(r_{out}/r_{in})]/k + 1/h_{out}$
- Note: $U_{in} * A_{in} = U_{out} * A_{out}$

Fouling Factor Concept

- Over time, surfaces accumulate deposits (rust, scale, algae).
- These deposits act as additional thermal resistance.
- We account for this using a 'fouling factor' (R_f).
- R_f has units of $\text{m}^2 \cdot \text{K} / \text{W}$.

Including Fouling in U

- We simply add the fouling resistances to our total resistance.
- $1/U = 1/h_{in} + R_{f,in} + L/k + R_{f,out} + 1/h_{out}$
- Fouling always decreases the overall heat transfer coefficient.
- Designers must overestimate U requirements to account for future fouling.

Typical Values of U (W/m^2K)

- Water to Water: 850 - 1700
- Water to Oil: 110 - 350
- Steam condenser (Water in tubes): 1000 - 6000
- Air to Water: 10 - 50
- Notice that systems involving gases (air) have much lower U values.

Example Problem

- A steel pipe ($k=50$) has $r_{in} = 20$ mm, $r_{out} = 24$ mm.
- Inside water $h_{in} = 1000$ W/m²K. Outside air $h_{out} = 15$ W/m²K.
- Calculate U based on the outside area (U_{out}).

Summary

- The overall heat transfer coefficient (U) combines conduction and convection.
- For cylinders, U must be referenced to a specific area (usually A_{in} or A_{out}).
- Fouling factors account for the thermal resistance of scale/dirt build-up.
- U simplifies the calculation of heat transfer in complex systems.

Next Lecture Preview

- Topic: Heat transfer from an extended surface (Fins)
- How can we increase heat transfer when convection is poor?
- We will explore the use of fins to increase surface area.